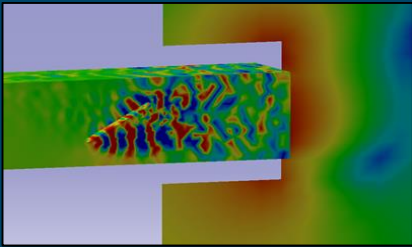


Better & faster vehicle NVH insights using the latest transfer path analysis methods

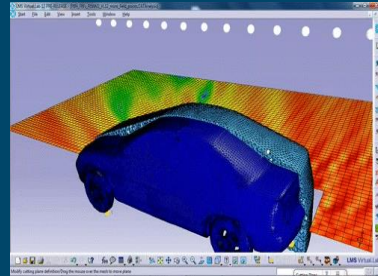
Webinar 20 November 2018

NVH Engineering Challenges

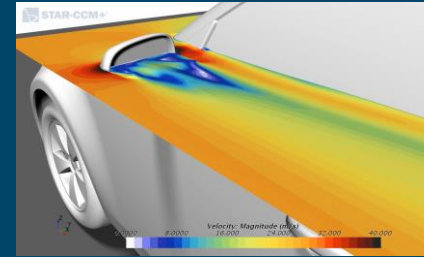
HVAC



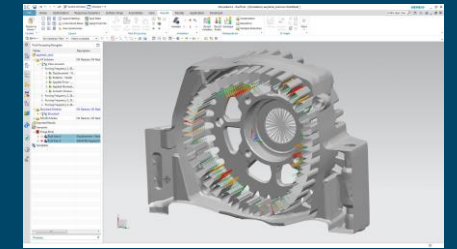
Pass-by noise



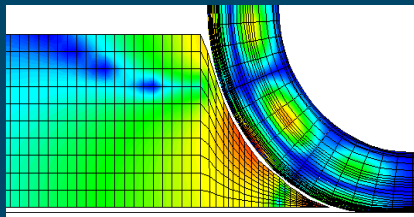
Wind noise



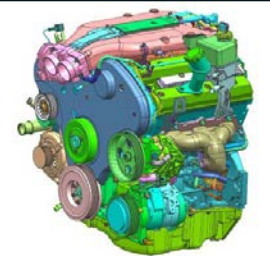
Electric motor



Road Noise



Engine



Better & faster vehicle NVH insights using the latest transfer path analysis methods

Introduction to transfer path analysis

Traditional TPA methods

Time-domain TPA

Component-based TPA

Model-based TPA

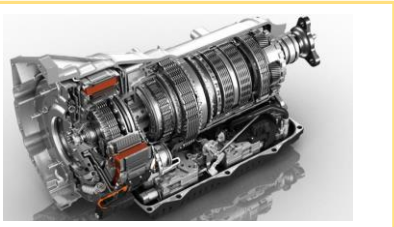
Conclusions



Introduction to Transfer Path Analysis

Source-transfer-receiver approach

Transmission



Drive line



HVAC



Exhaust



Wiper System



EPS



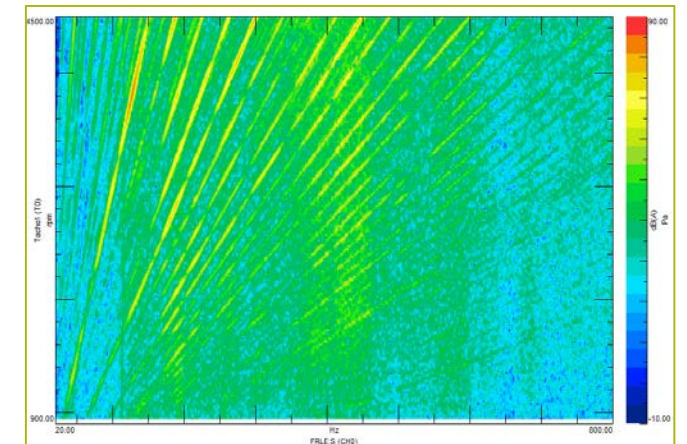
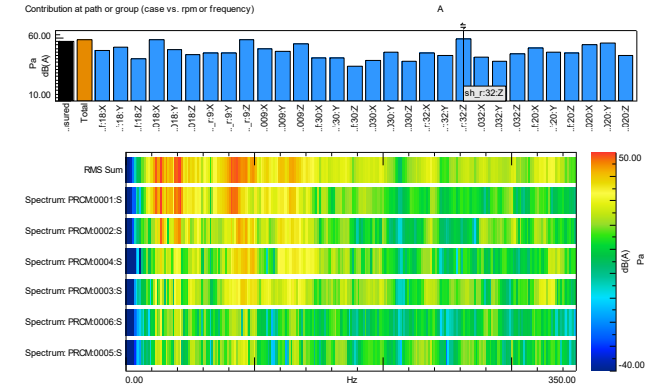
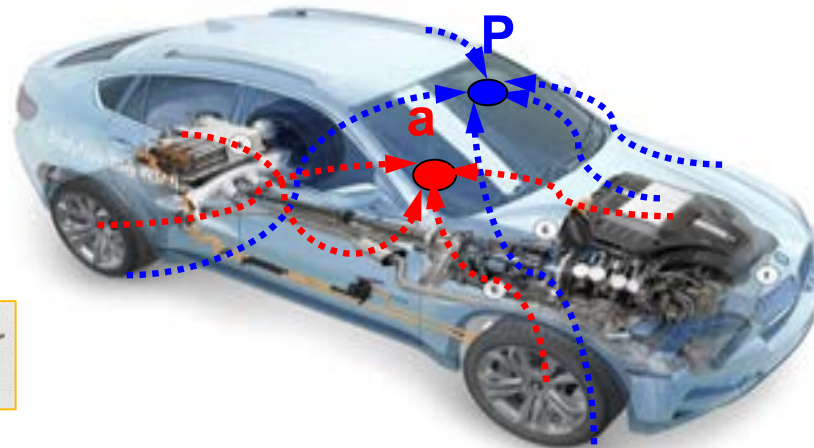
(H)EV



Tyres



Engine



Source (F_i, Q_j)

X

Transfer (NTF)

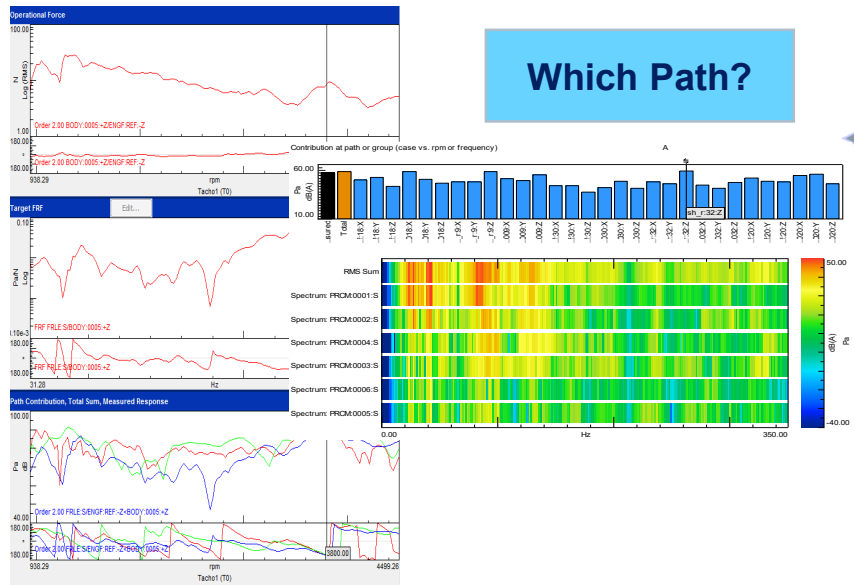
=

Receiver (y_k)

Introduction to Transfer Path Analysis

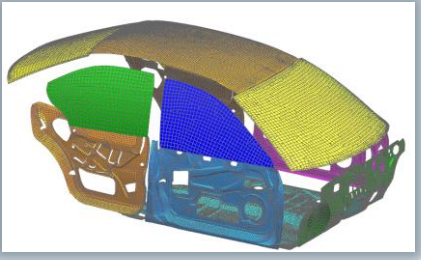
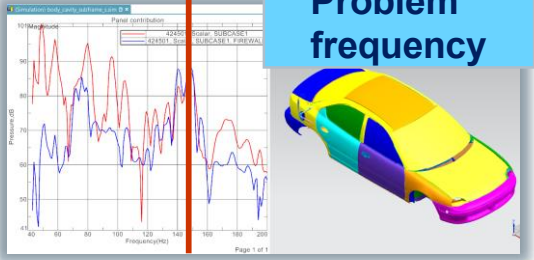
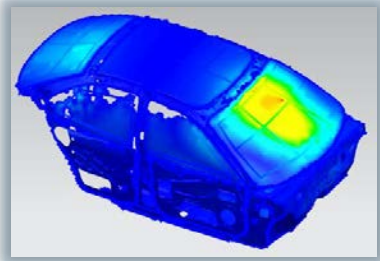
TPA as part of Contribution Analysis

Different ways of looking at a NVH problem

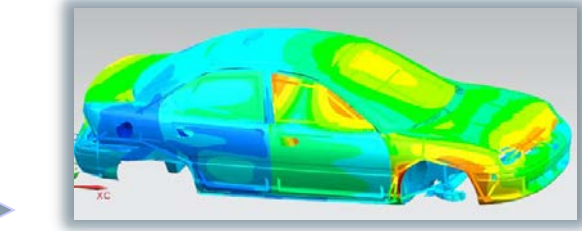


Which Path?

Which grid?



Which Panel?



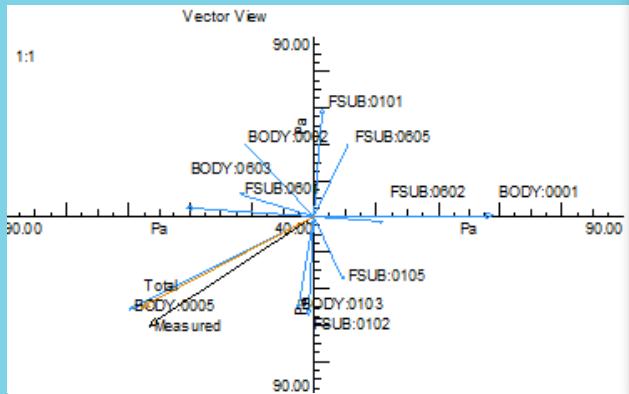
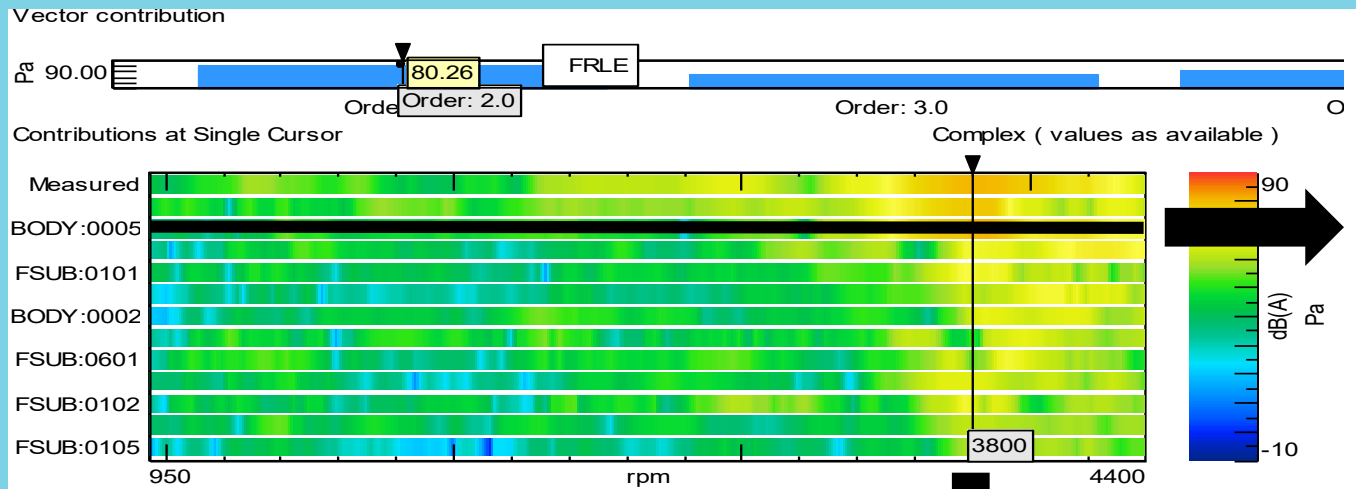
What operating shape?

Introduction to Transfer Path Analysis

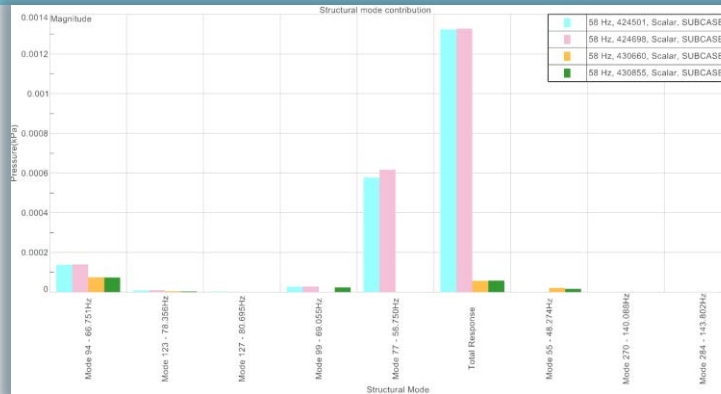
TPA as part of Contribution Analysis

Step 1: Contribution Analysis

Paths



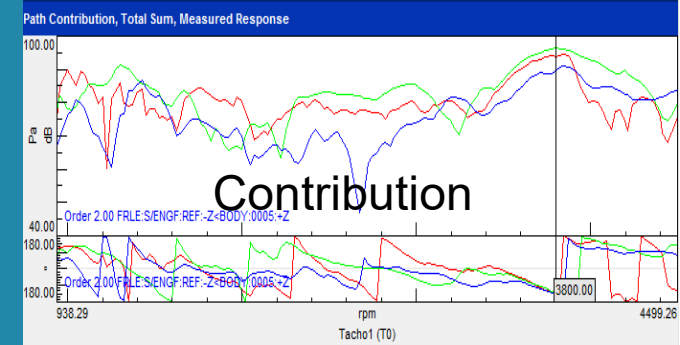
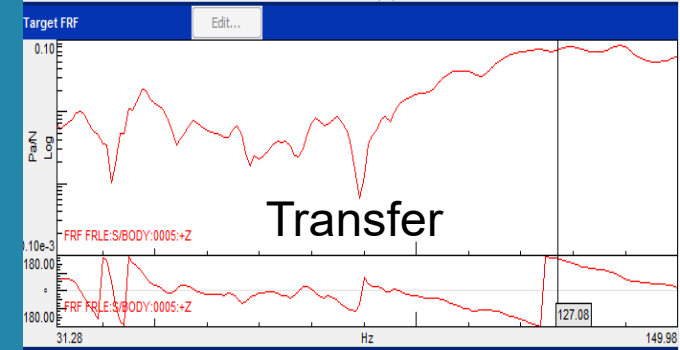
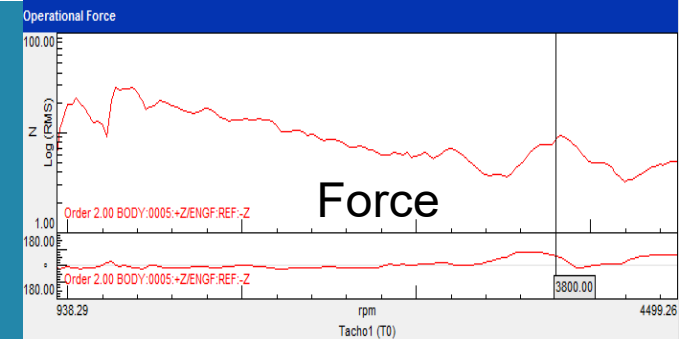
Amplitude Phase



Modal

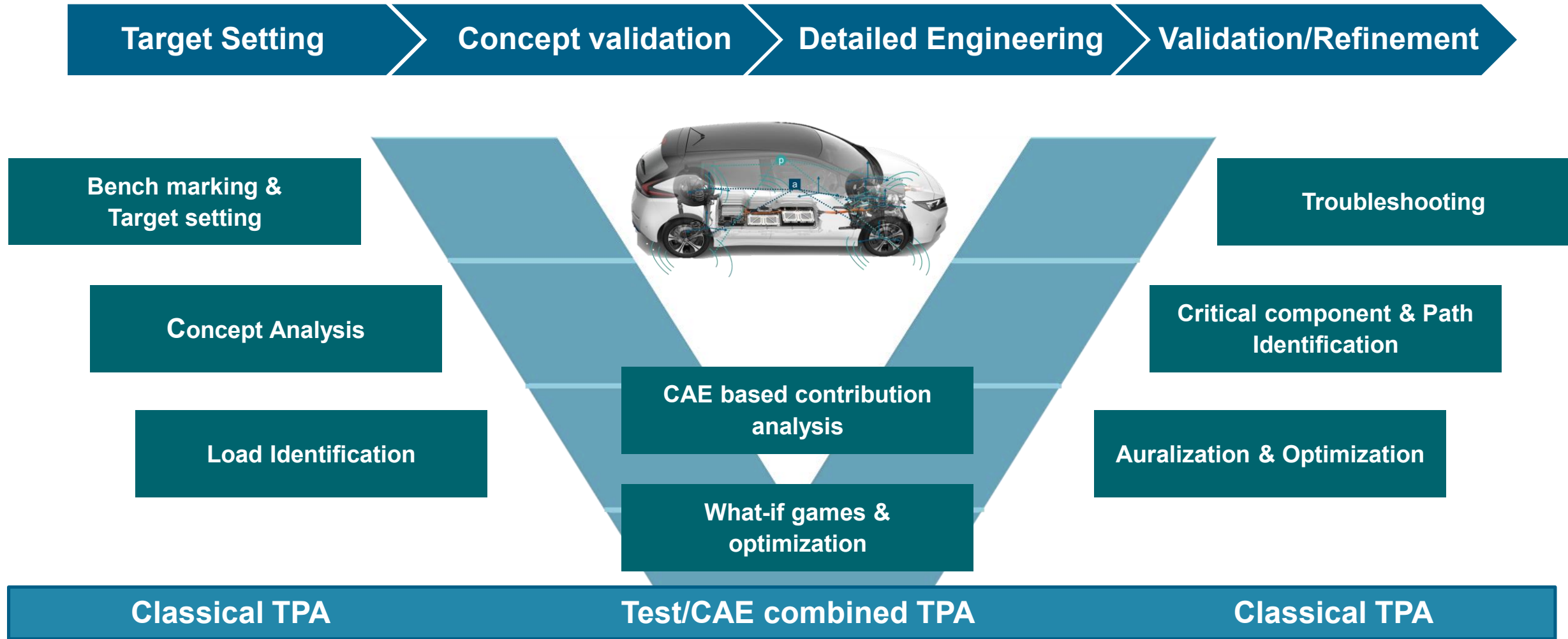
Step 2:

Path Source Transfer



Transfer Path Analysis

Throughout the vehicle development cycle



Better & faster vehicle NVH insights using the latest transfer path analysis methods

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Conclusions



Transfer Path Analysis

Source-transfer-receiver approach

Transmission



Drive line



HVAC



Exhaust



Wiper System



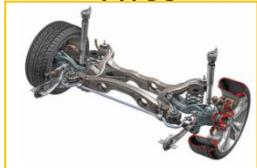
EPS



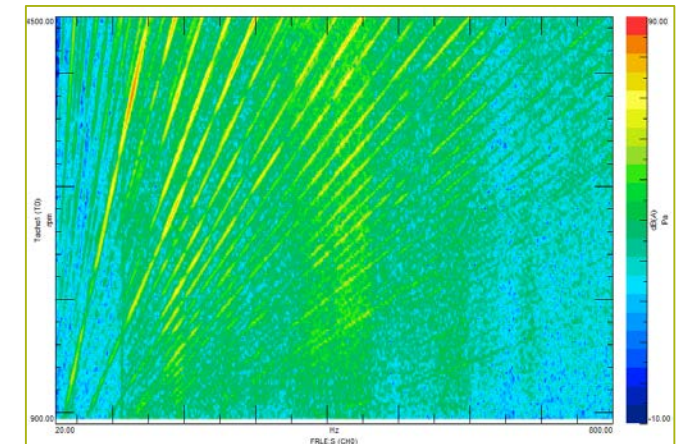
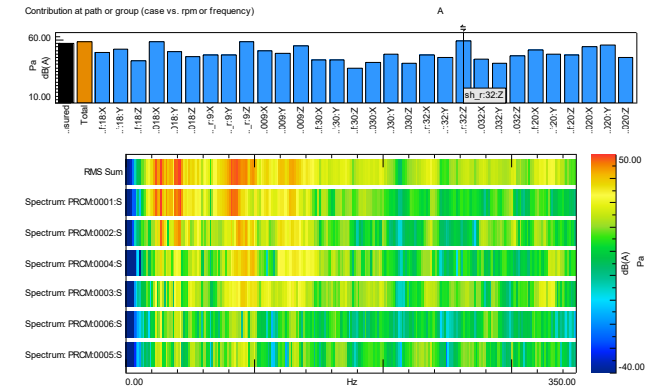
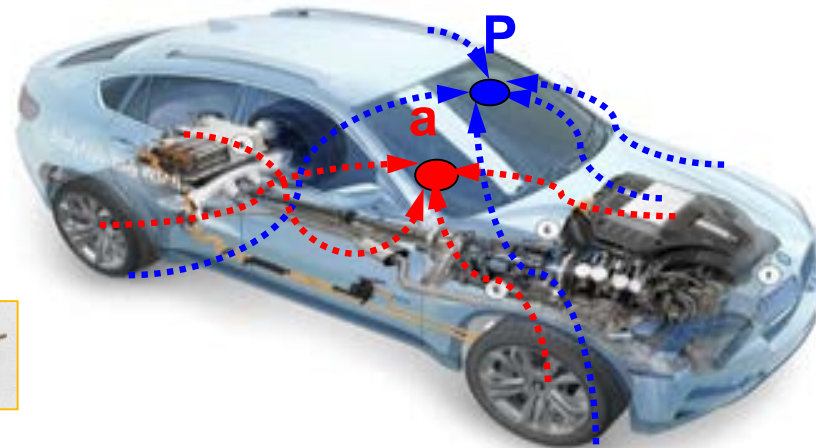
(H)EV



Tyres



Engine



Source (F_i, Q_j)

X

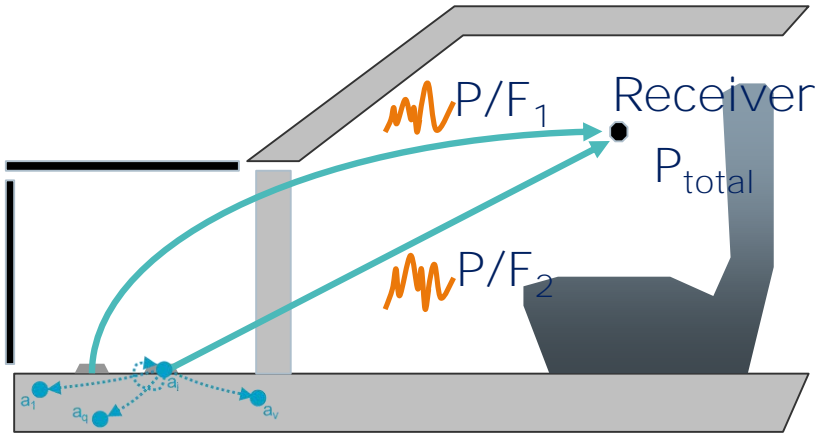
Transfer (NTF)

=

Receiver (y_k)

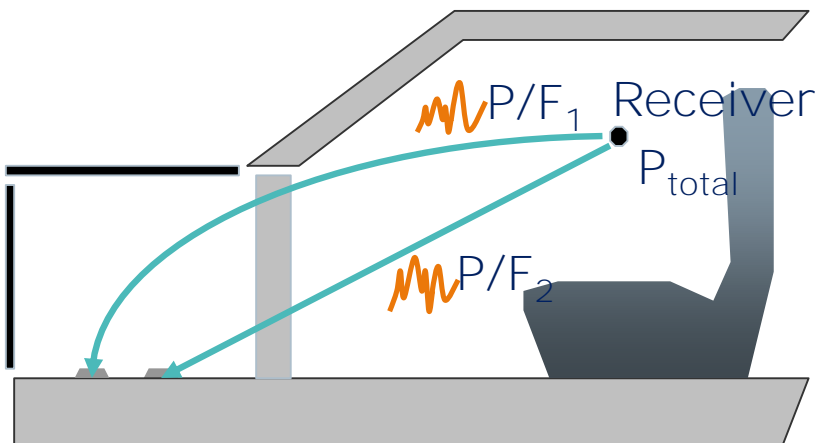
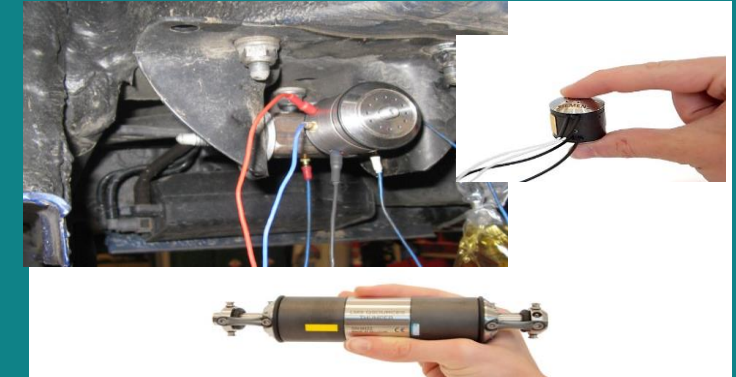
Traditional TPA methods

Efficient & Accurate FRF Acquisition



Direct FRF Measurements

- Modal Hammer
 - ✓ Common approach but prone to measurement and repeatability error
- Shaker
 - ✓ Miniaturized shakers can overcome many hammer issues



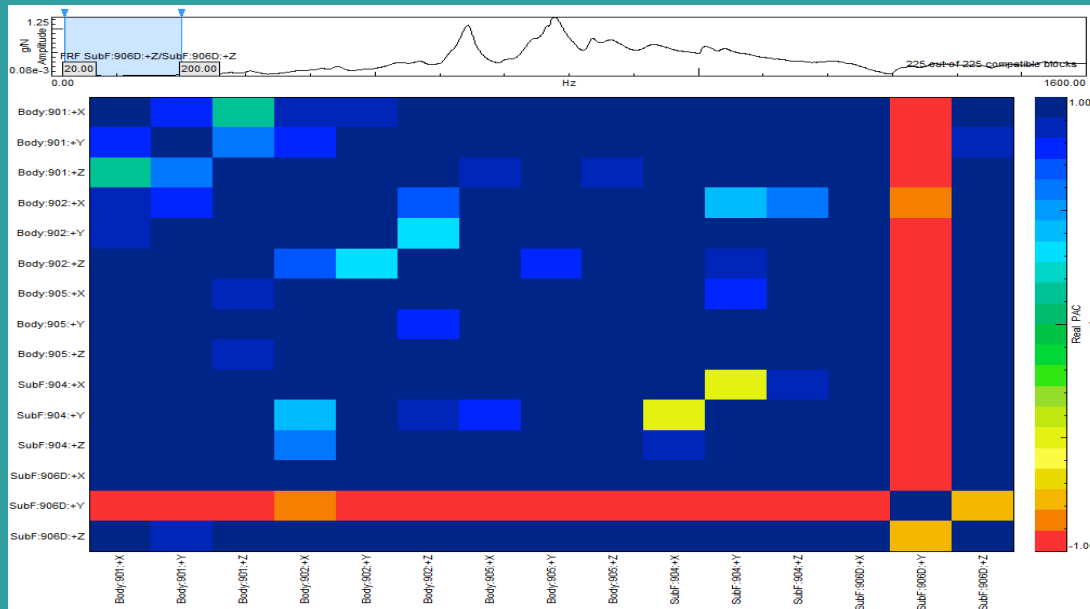
Reciprocal FRF measurement

- Measurement of multiple/all NTF (noise transfer functions) at once rather than roving hammer or shakers

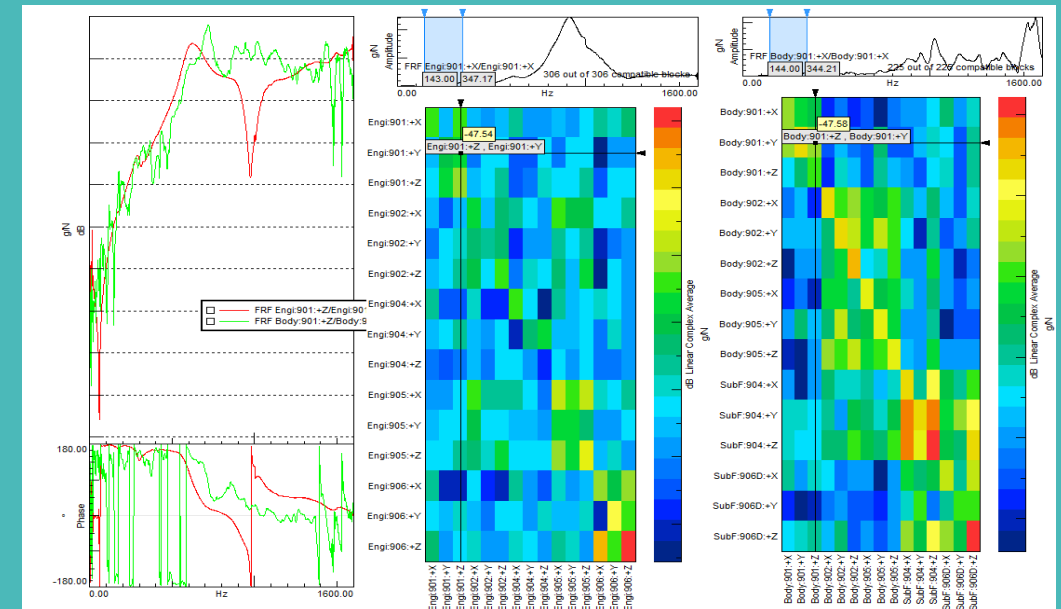


Traditional TPA methods

Requirements: Verify quality data quality and gain insight



Instant verification of FRF consistency by checking reciprocity, linearity and directions errors

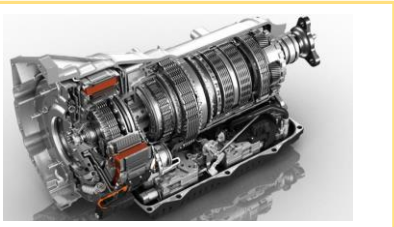


Gain insight in the system dynamics
1000+ FRFs in one view

Transfer Path Analysis

Source-transfer-receiver approach

Transmission



Drive line



HVAC



Exhaust



Wiper System



EPS



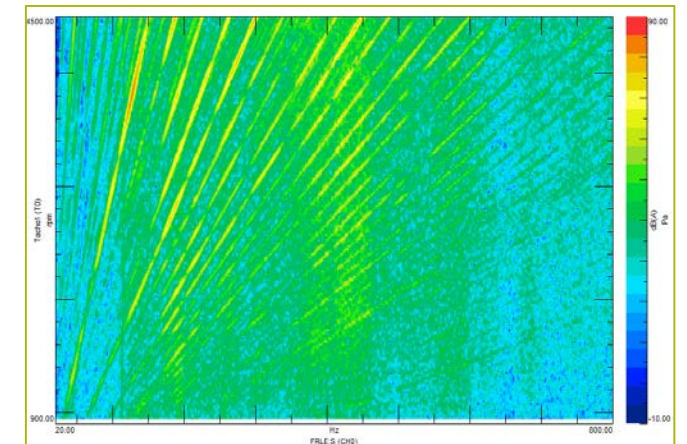
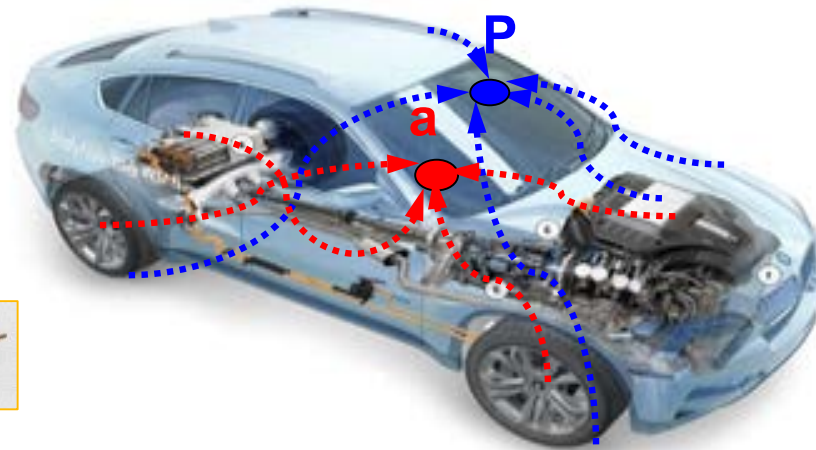
(H)EV



Tyres



Engine



Source (F_i, Q_j)

X

Transfer (NTF)

=

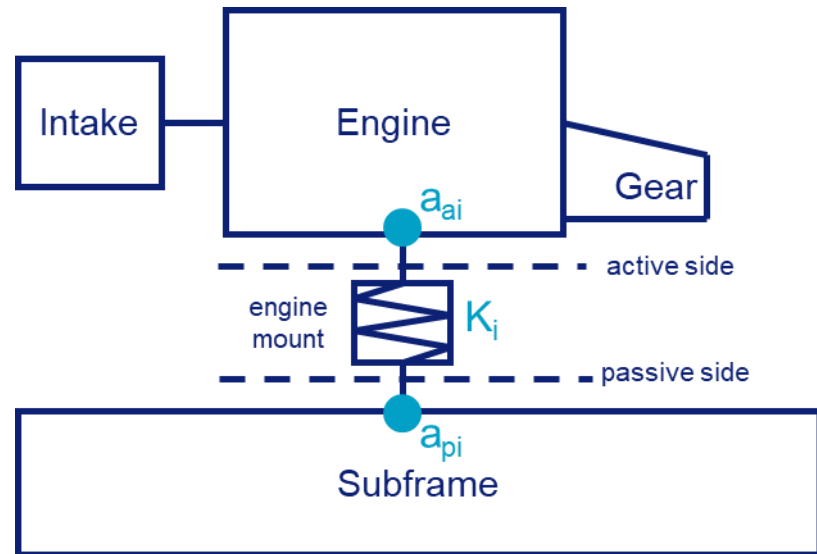
Receiver (y_k)

Load identification methods

Mount stiffness and Matrix inversion

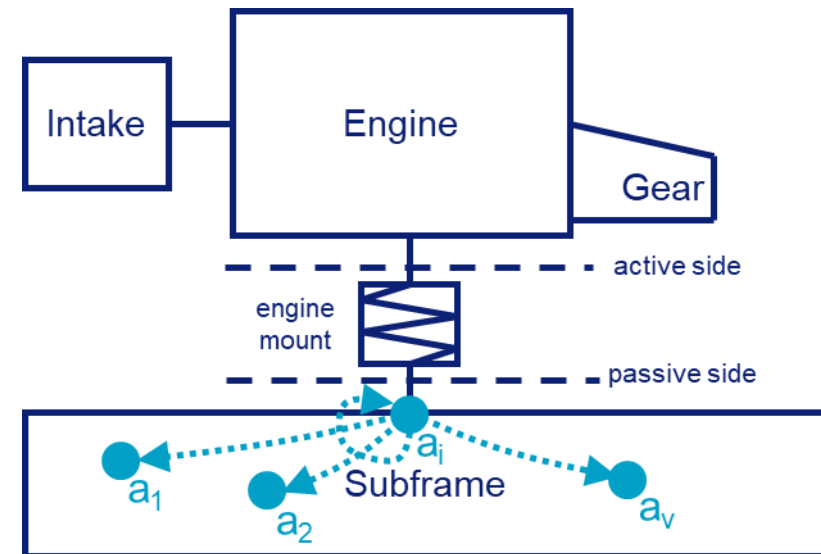
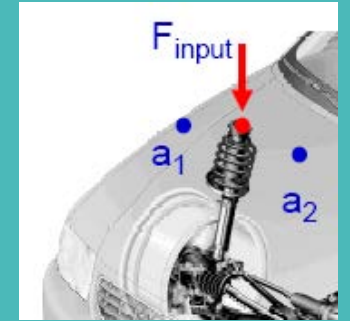
Mount Stiffness

$$F_i(\omega) = K_i(\omega) \cdot \frac{(a_{ai}(\omega) - a_{pi}(\omega))}{-\omega^2}$$



Matrix Inversion

$$\{F(\omega)\} = [H(\omega)]^{-1} \cdot \{a(\omega)\}$$



Load identification methods

Mount stiffness and Matrix inversion

Mount Stiffness

$$F_i(\omega) = K_i(\omega) \cdot \frac{(a_{ai}(\omega) - a_{pi}(\omega))}{-\omega^2}$$



Requires

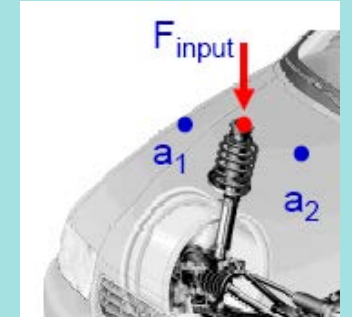
- FRFs to targets only
- Active & Passive side responses
- Mount Stiffness Curves
- Enough isolation over mount (~20dB)

(dis)advantages

- Only target-FRFs required
- Insight in mount-isolation performances
- Disassembly into trimmed-body condition
- Requires accurate mount-stiffness curves

Matrix Inversion

$$\{F(\omega)\} = [H(\omega)]^{-1} \cdot \{a(\omega)\}$$



Requires

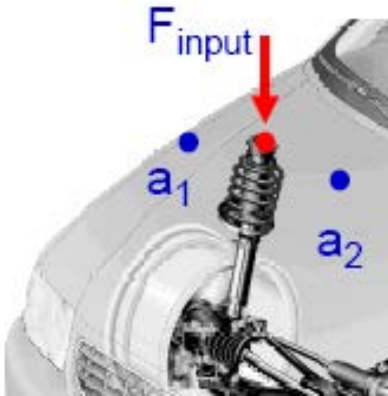
- FRFs to targets & indicators
- Full structural FRF matrix
- Only Passive side responses
- Enough over-determination (>2x)

(dis)advantages

- No Mount-stiffness curves required
- Detailed body information
- Disassembly into trimmed-body condition
- Time-consuming FRF measurement (Full Matrix)
- No separation of nearby paths

Traditional TPA method: Advanced load identification

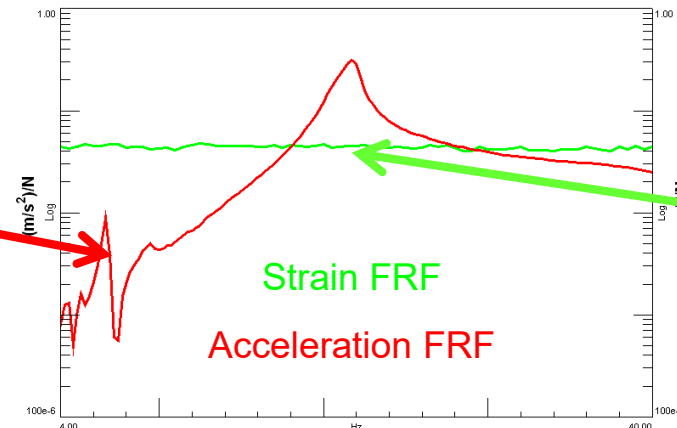
Strain sensors: separating nearby paths



$$\{F(\omega)\} = [H(\omega)]^{-1} \cdot \{a(\omega)\}$$

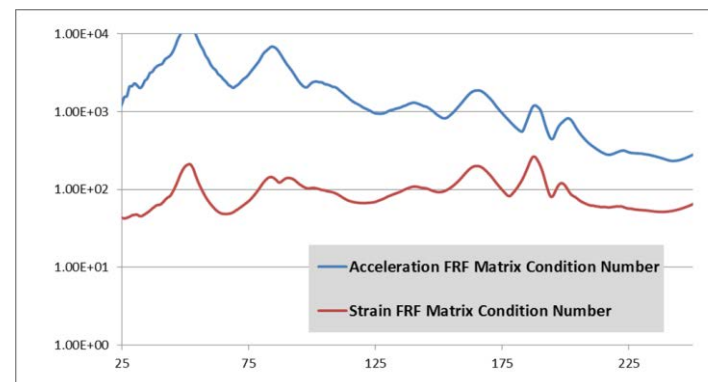
Classic approach: Acceleration Indicators

- Acceleration responses are dominated by a **limited** amount of **global** structural body modes
- Potential limitation for force estimation possibilities



Advanced approach: Strain Indicators

- Strain responses are dominated by a **high** amount of **local** structural body modes
- Extended possibilities for load separation / identification



Load identification

Simcenter Testlab OPAX

$$P(\omega) = \sum H_i(\omega) F_i(\text{parameters}, a_{ai}(\omega), a_{pi}(\omega))$$

- Soft mounts $F_i(\omega) = \underline{K_i} \frac{(a_{ai}(\omega) - a_{pi}(\omega))}{-\omega^2}$
- Hard mounts $F_i(\omega) = \underline{K_i} \frac{a_{ai}(\omega)}{-\omega^2}$

Fast method applying force estimation method using parametric model to reduce required measurements

Requires

- FRFs to targets (structural FRFs are optional)
- Active & Passive side responses
- Acceleration (or Strain) indicators

(dis)advantages

- Limited set of FRFs required
- No disassembly into trimmed-body condition (depends on the expected accuracy)
- Limited body-information (compared to Matrix-Inversion)

Better & faster vehicle NVH insights using the latest transfer path analysis methods

Introduction to transfer path analysis

Traditional TPA methods

Time-domain TPA

Component-based TPA

Model-based TPA

Conclusions



Frequency-domain vs. Time-domain TPA

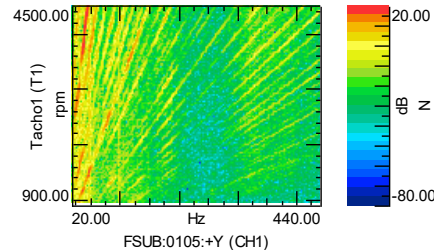
Frequency-domain TPA

- ➔ Order analysis
- ➔ Spectrum analysis
- ✓ Run-up & run-down
- ✓ Stationary: e.g road noise

Time-domain TPA

- ➔ Time traces
- ✓ Run-up & run-down
- ✓ Stationary: e.g road noise
- ✓ Transient: e.g. engine start-up
- ✓ Semi-stationary: e.g. idle noise, frequency modulation ...

Loads (orders, spectra)

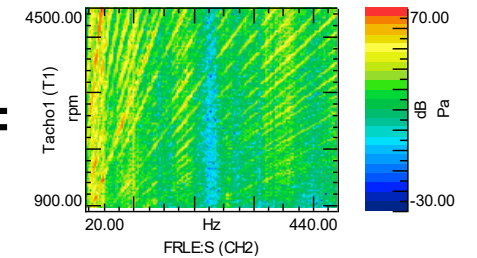


X

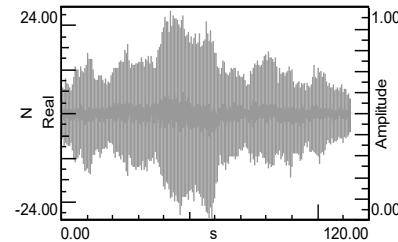
NTF

Frequency transfer model

Path contributions (orders, spectra)

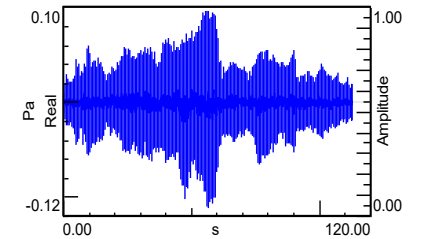


=



⊗

FIR Filter



=

Loads (time traces)

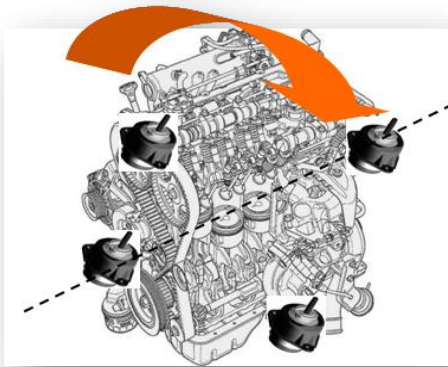


Path contributions (time traces)

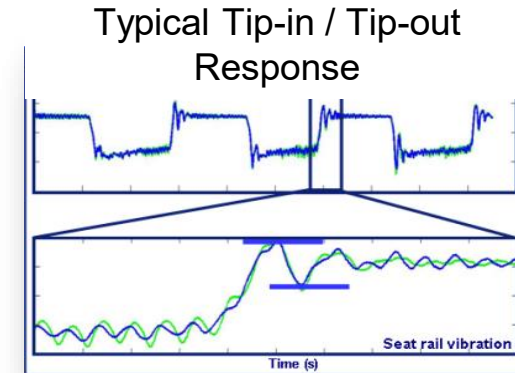


Auralization, Signature Analysis, Sound Quality metrics ...

Time-domain TPA for Tip-in/Tip-out Application Example



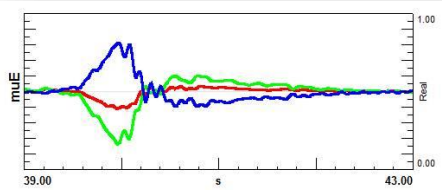
Transient torque oscillations in the driveline
Amplified by P/T, suspension and body modes
Resulting seat and steering-wheel vibrations



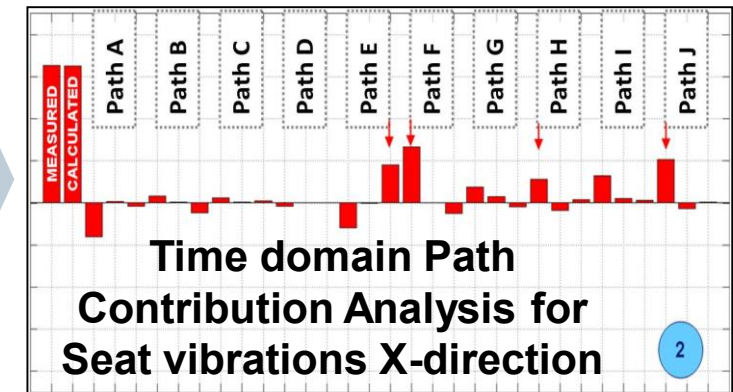
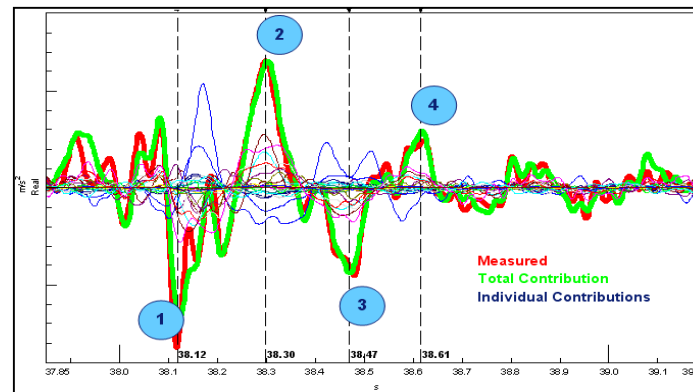
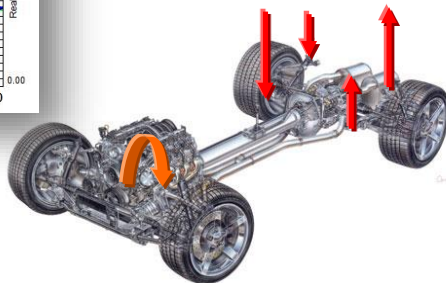
Transient Load identification
Strain gage technology
Low frequency

...in "full NVH Context"
To link transient loads
to Dynamic Interface Forces...

...Transfer Path Analysis (TPA)
Insights in contribution
to NVH Comfort



Strain responses
(right) during Tip-in



Better & faster vehicle NVH insights using the latest transfer path analysis methods

SIEMENS
Ingenuity for life

Introduction to transfer path analysis

Traditional TPA methods

Time-domain TPA

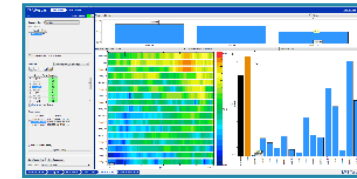
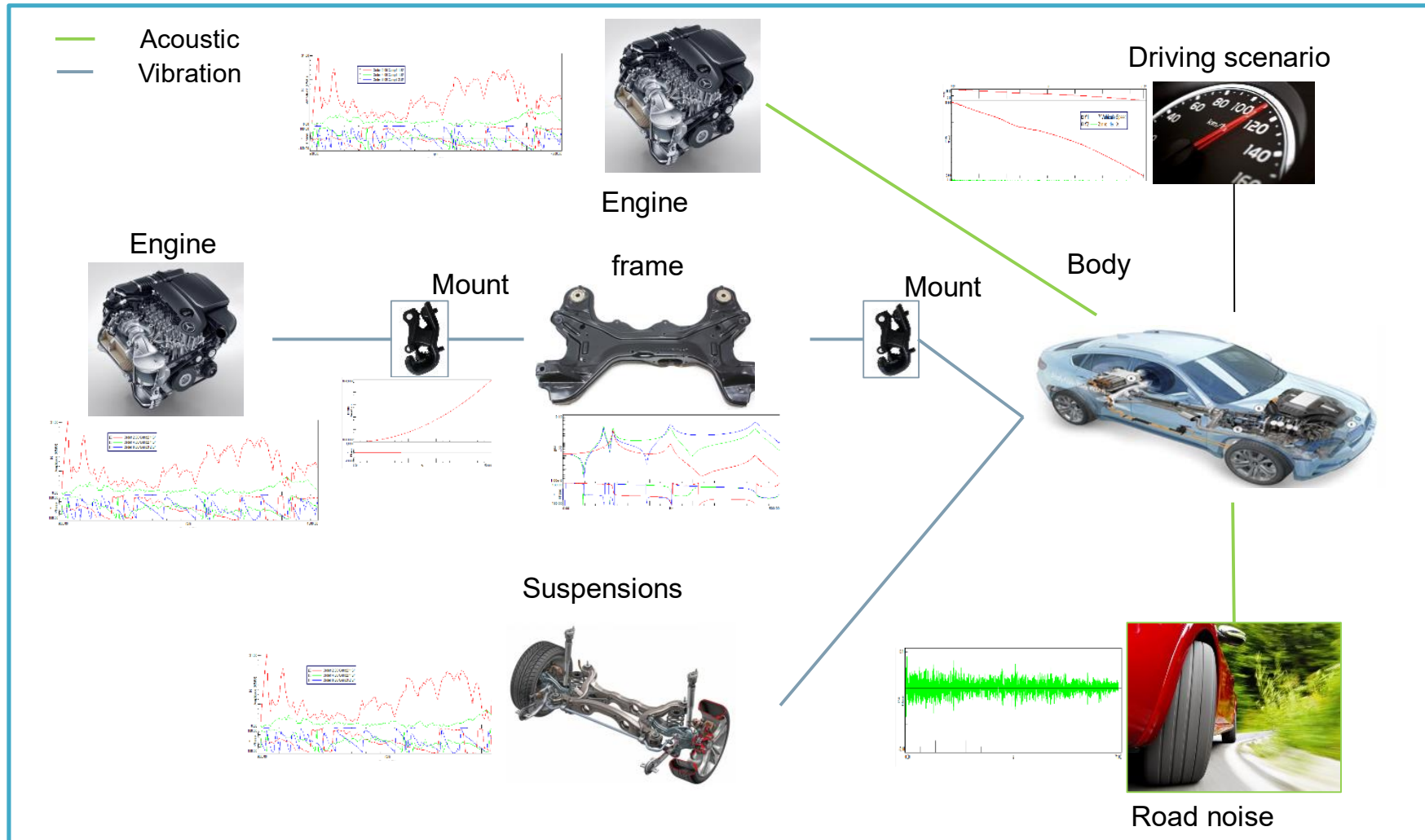
Component-based TPA

Model-based TPA

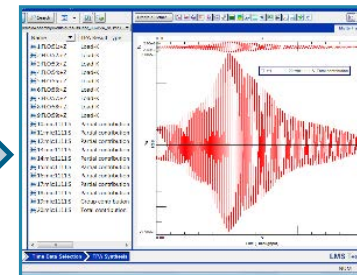
Conclusions



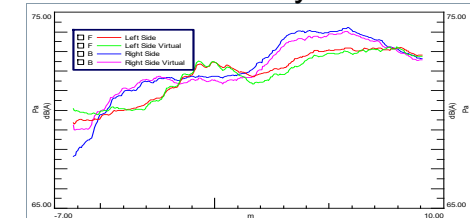
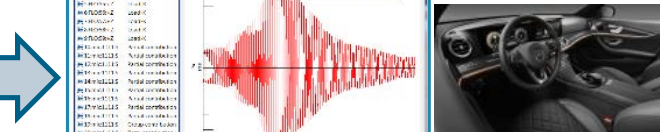
Component-based TPA Concept



Compare Contribution Analysis

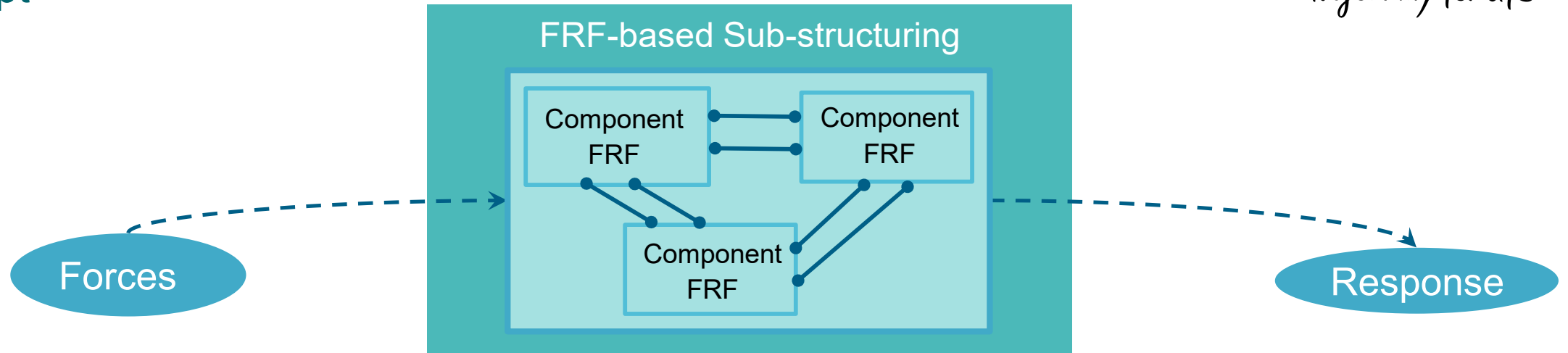


Sound Synthesis Result SQ Analysis

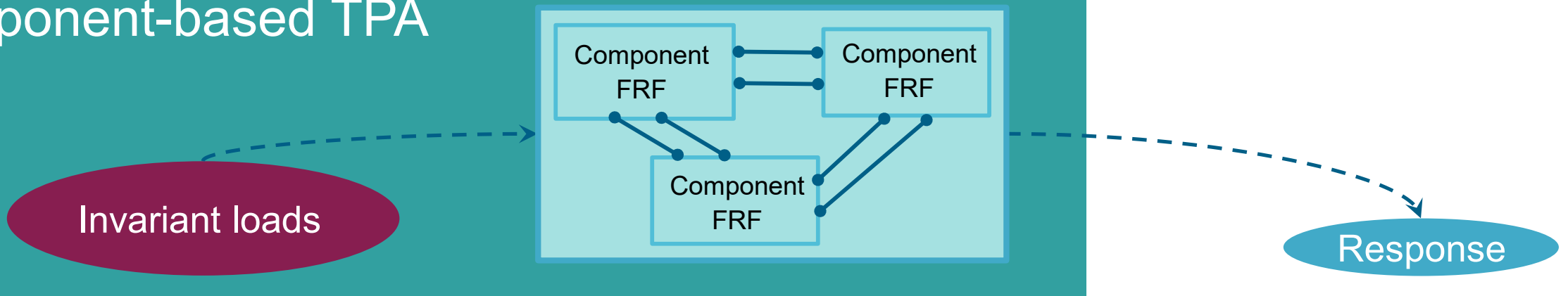


Pass-by Noise Synthesis

Component-based TPA Concept



Component-based TPA



Source (F_i, Q_j)

X

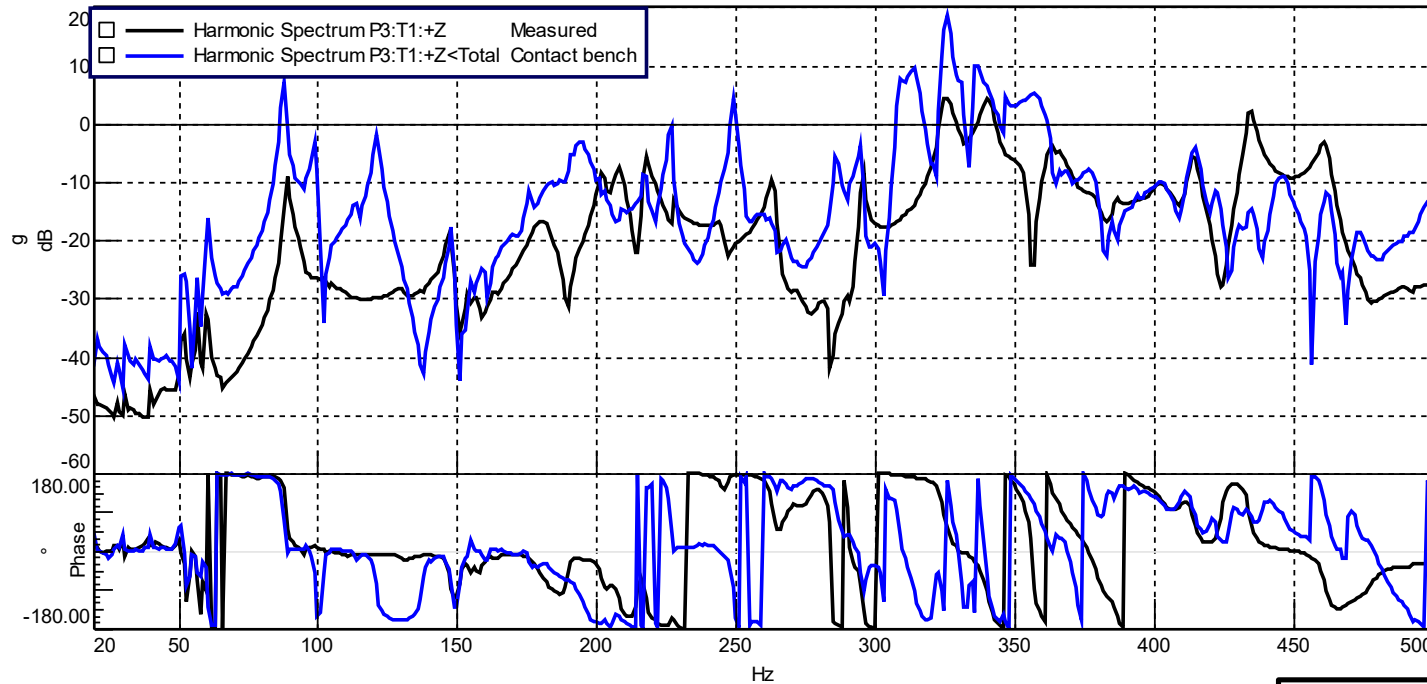
Transfer (NTF)

=

Receiver (y_k)

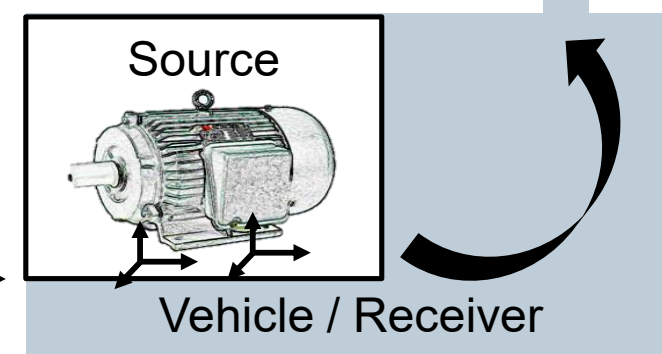
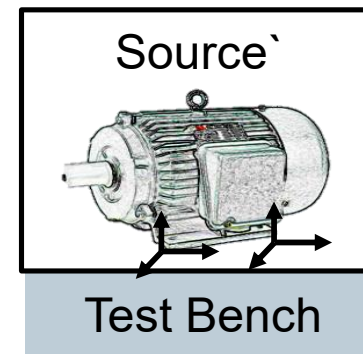
Example: Source-Receiver interaction

Strong coupling case – Structure Borne



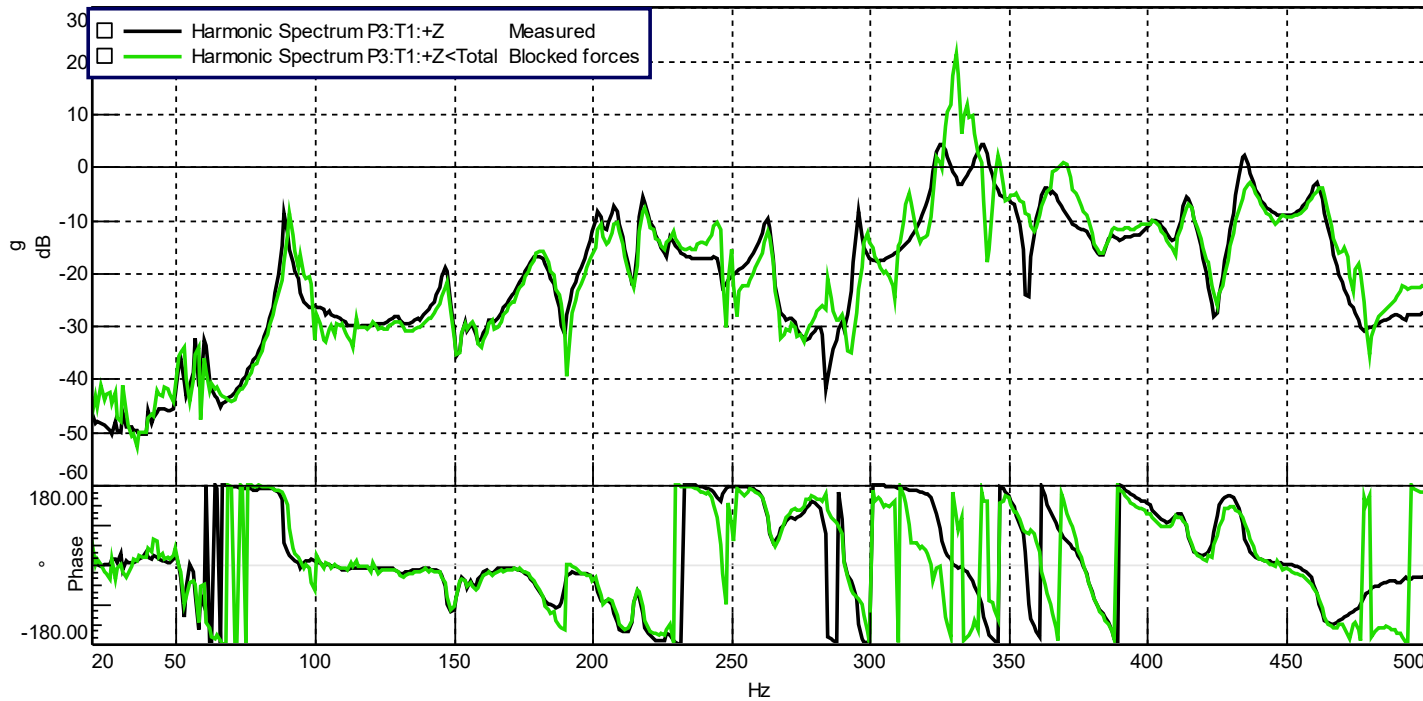
~~Exchange Contact Forces~~

Predict Target Response



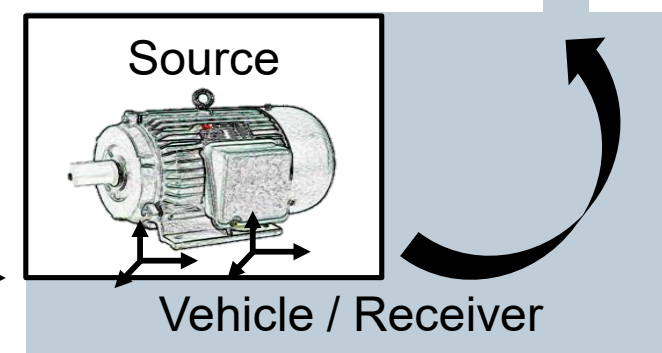
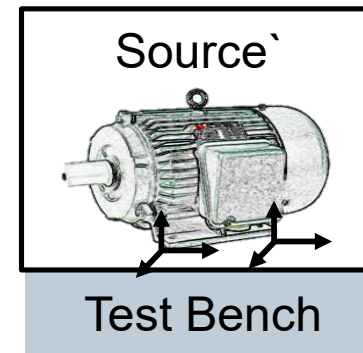
Example: Source-Receiver interaction

Strong coupling case – Structure Borne



Exchange Blocked Force
Invariant Source Description

Predict Target Response

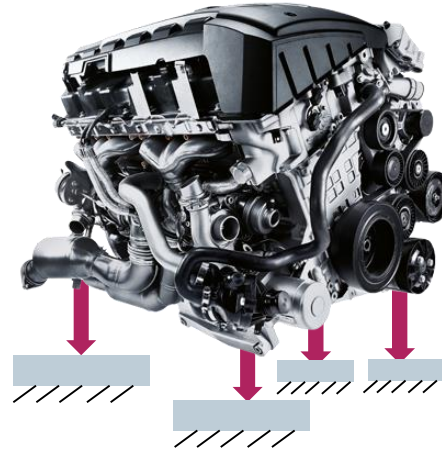


Component-based TPA

Invariant load characterization

Structure-borne:

Blocked Forces



Airborne:

Volume Velocities



Invariant load characterization

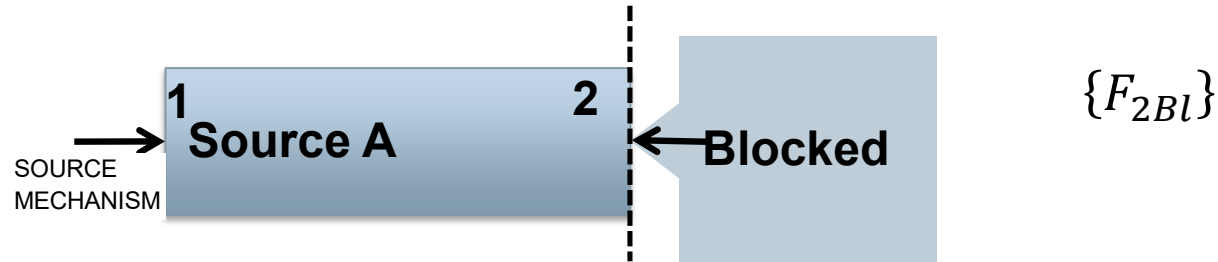
Receiver independent, allowing:

- Validating sources against receiver independent targets
- Benchmarking or validating modifications
- Predicting NVH performance in arbitrary source-receiver assemblies

Component-based approach

Structure-borne: Blocked Forces

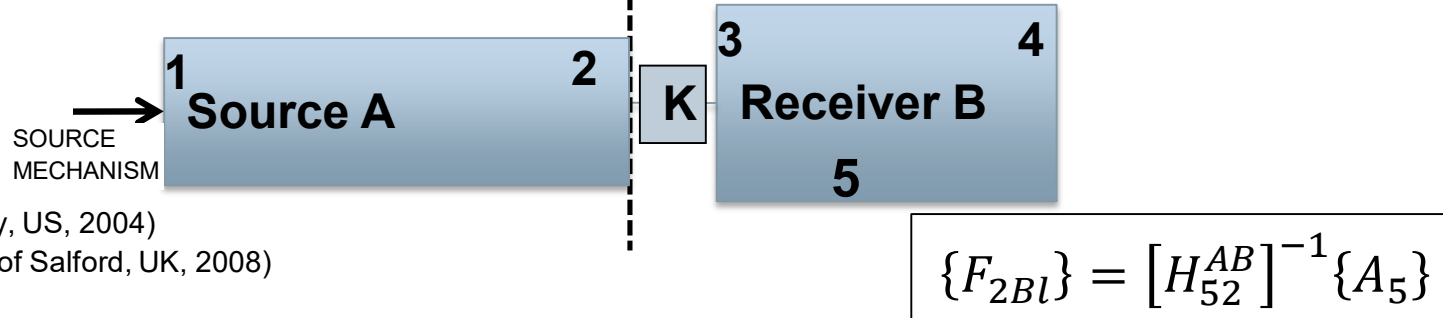
1. Blocked Force



2. Free Velocity



3. In-Situ TPA

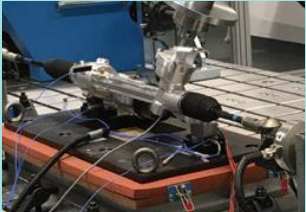
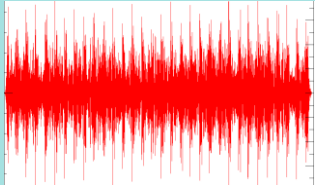




Calculate Interface Force using FBS

(Park / Gu, Ford Motor Company, US, 2004)
(Moorhouse & Elliott, University of Salford, UK, 2008)

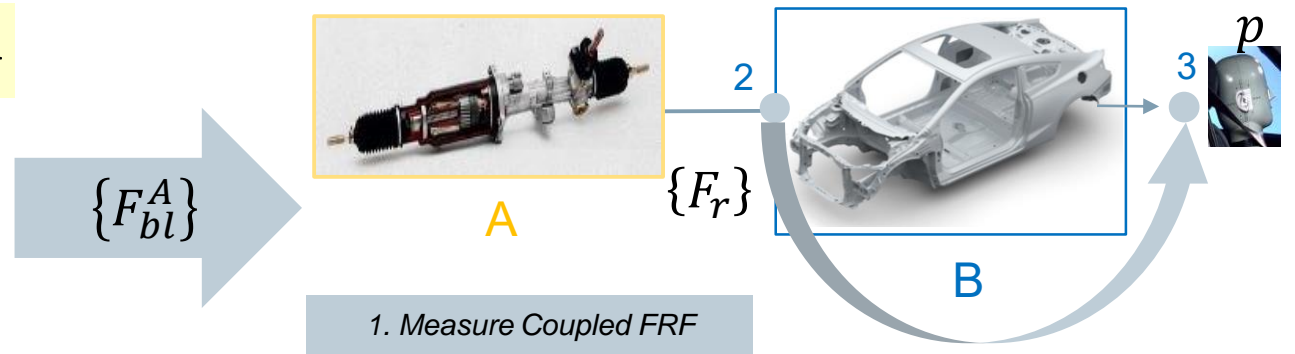
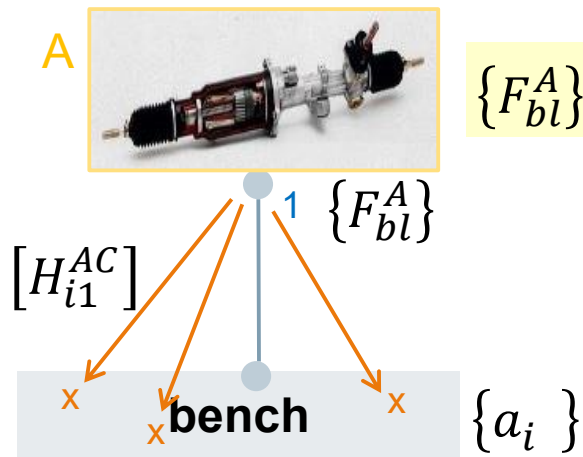
Component-based TPA

Application example of a steering system

Source Mechanism	Invariant Source Synth. Model	Sub-Receiver	Receiver
 <p>Steering System</p>	 <p>Blocked Forces & Impedances Mount Pos.</p>	 <p>Subframe FEM/TEST FRF</p>	 <p>Body FEM/TEST FRF</p>

Invariant Source Load (TEST)

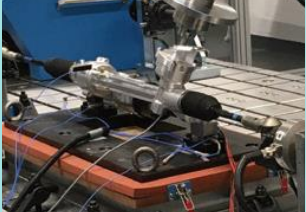
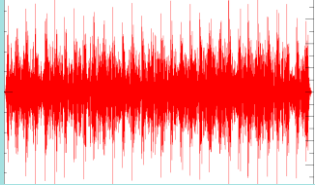


Full System Transfer Function



$$p = [H_{32}^{AB}] * \{F_{bl}^A\}$$

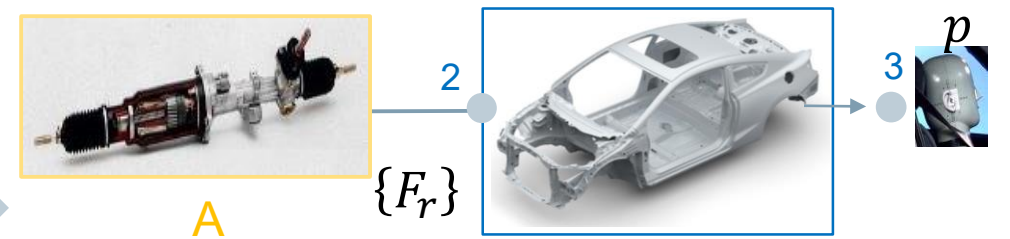
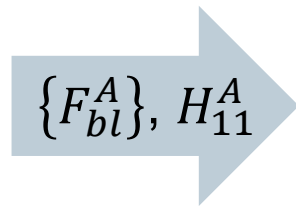
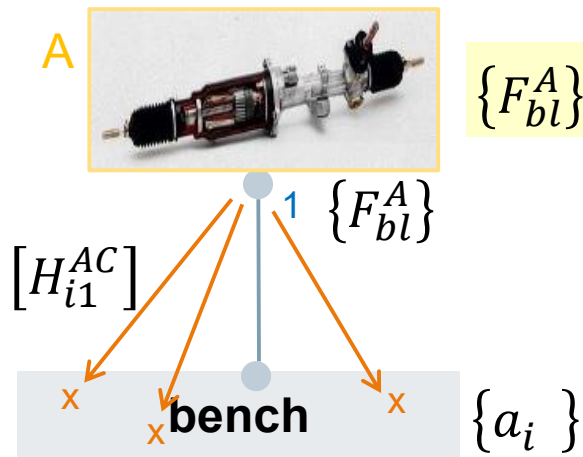
Component-based TPA

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Invariant Source Load (TEST)

Full System Transfer Function



2. Calculate Coupled FRF

$$\{F_r\} = [H_{11}^A + H_{22}^B + K^{-1}]^{-1} * [H_{11}^A] * \{F_{bl}^A\}$$

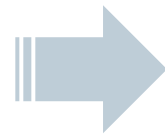
$$p = [H_{32}^B] * \{F_r\}$$

Component-based TPA

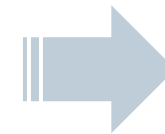
Application example of predicting Air-borne pass-by noise performance



Invariant description
of sources

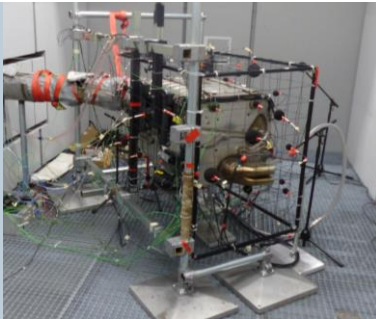


Recombine with selected
components to synthesize
SYSTEMS

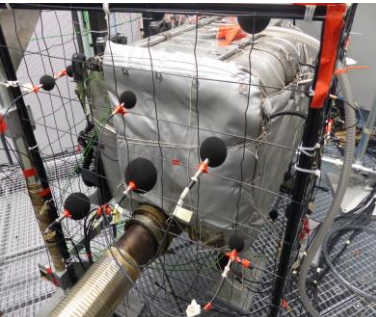


Predict NVH
performance

VARIANT
A



VARIANT
B



All Design Variants

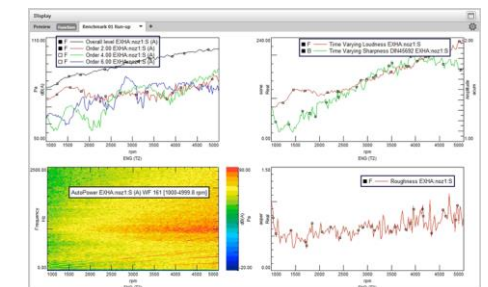
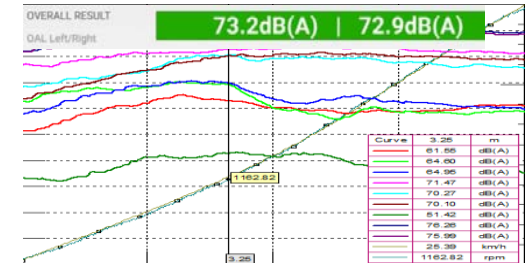
VEHICLE
1



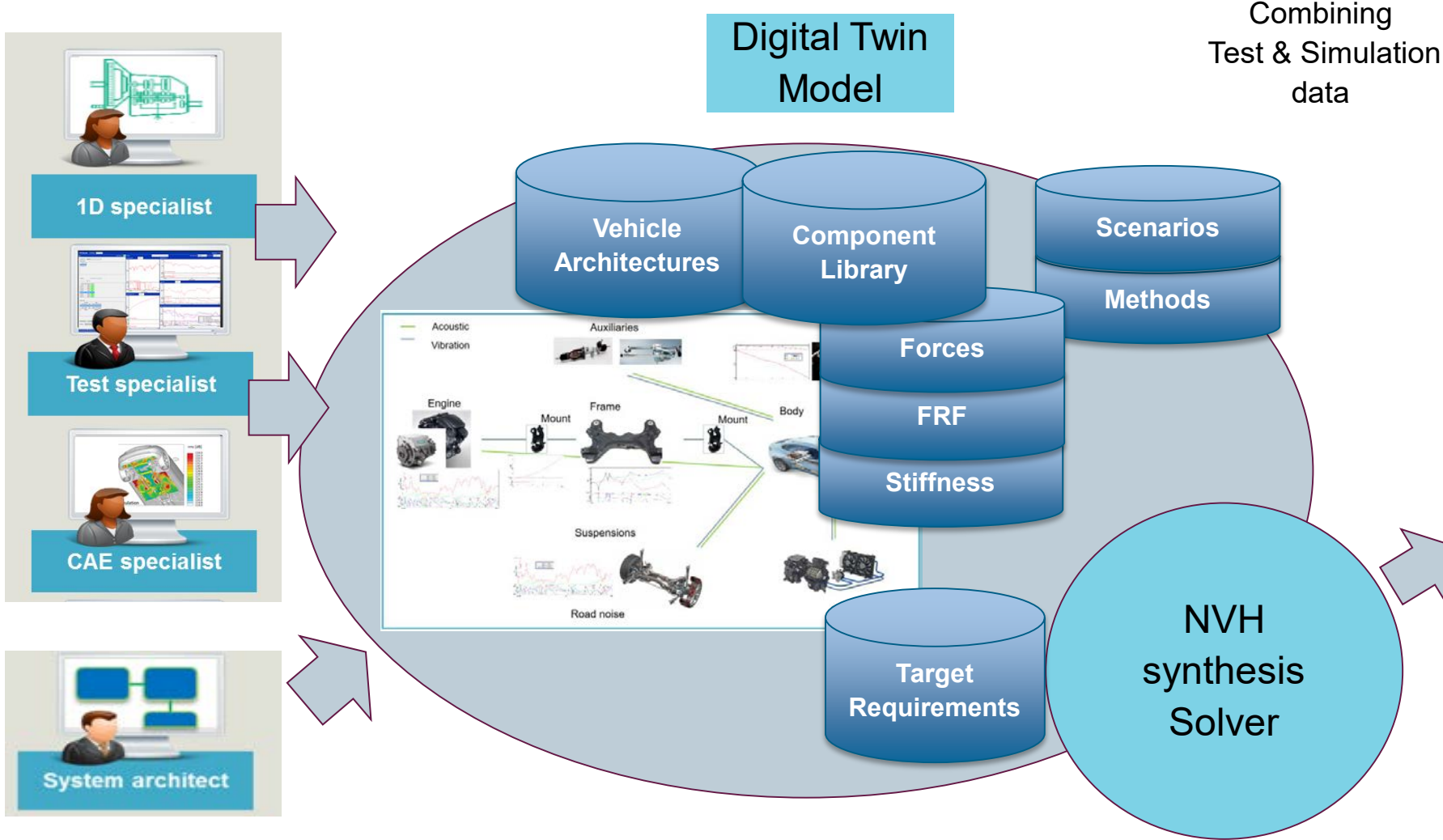
VEHICLE
VARIANTS



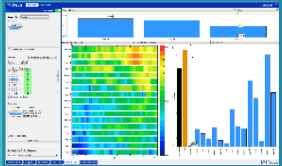
Vehicle Variants



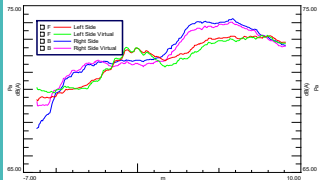
Component-based TPA System Engineering for NVH



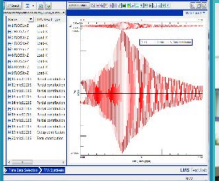
Compare Contribution Analysis




Pass-by Noise Synthesis



Sound Synthesis Result



NVH Driving Simulator Evaluation



Better & faster vehicle NVH insights using the latest transfer path analysis methods

Introduction to transfer path analysis

Traditional TPA methods

Time-domain TPA

Component-based TPA

Model-based TPA

Conclusions

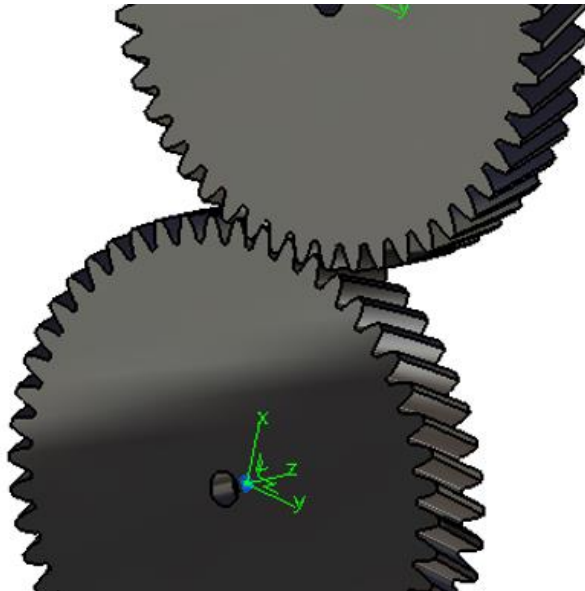


Model-base TPA

Gearbox Noise – Multi-Body Dynamics + Structural Dynamics

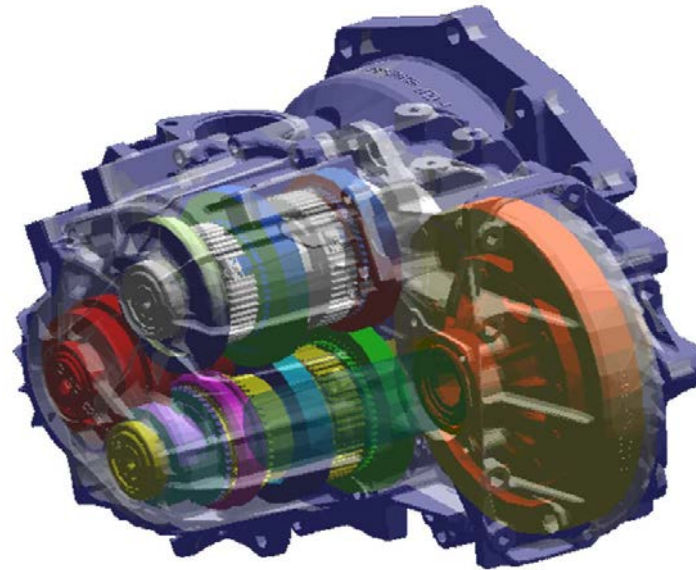
SOURCE

Gear rattle or meshing forces in function of volute profile, misalignment, torque load, ...



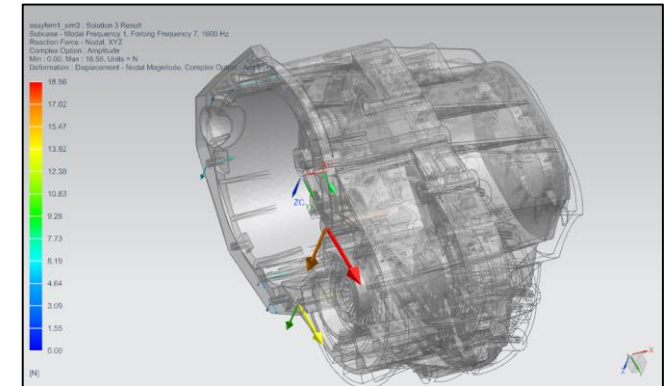
PATH TRANSFER

Full system of gears, axles and body including their respective flexibility

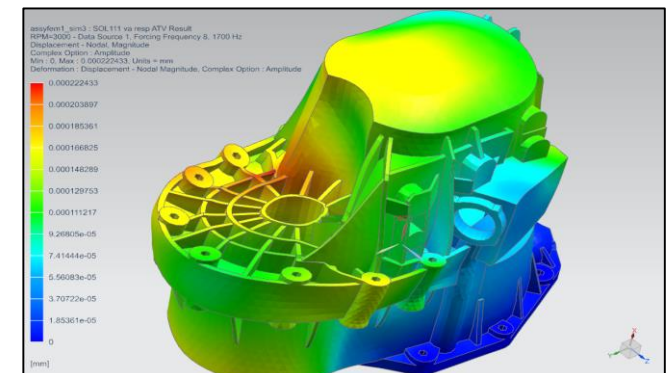


RECEIVER

Operational connection forces (to body)



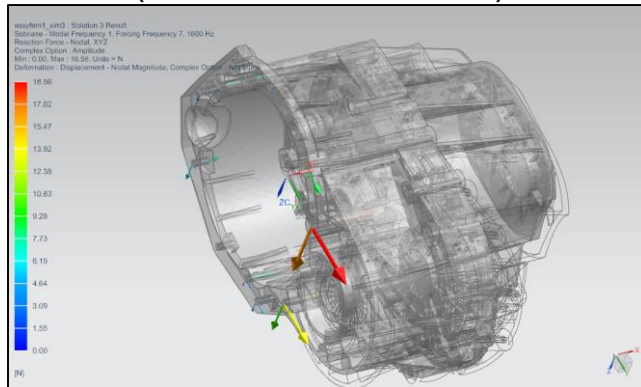
Operational surface vibrations



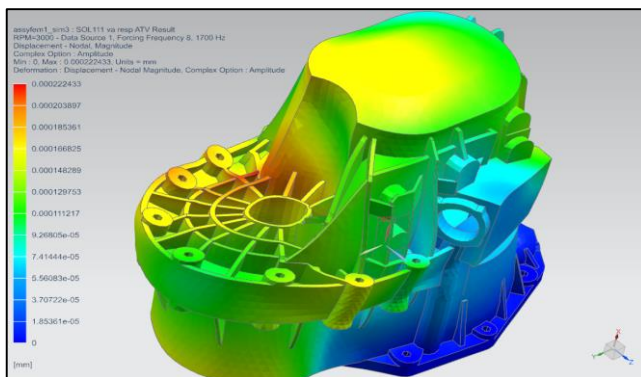
Gearbox Noise – Structural Dynamics and Acoustics

SOURCE

(structure borne noise)

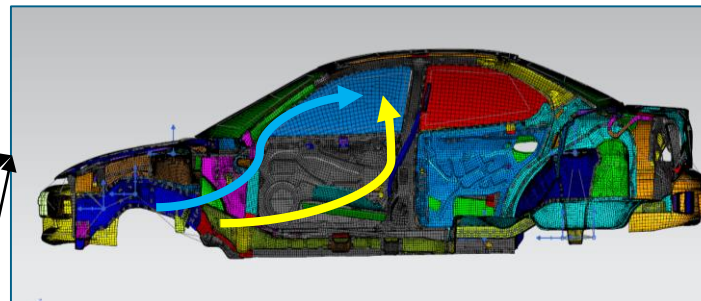


Operational surface vibrations
(Airborne noise)

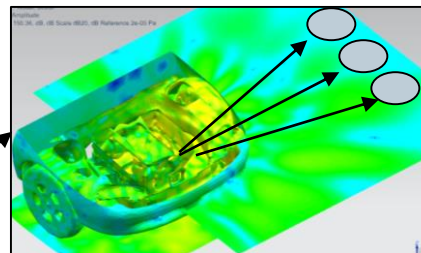


PATH TRANSFER

Vibro-Acoustic FRFs

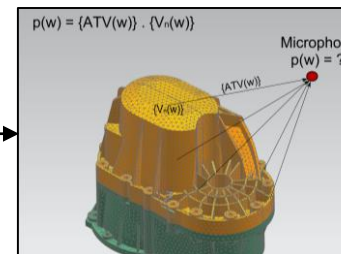


Acoustic FRFs SPL/ surface vibration



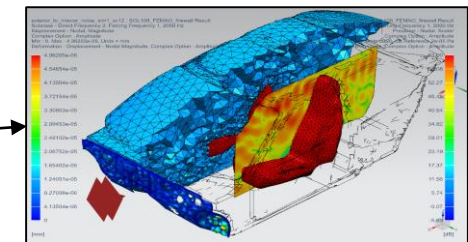
Installed condition

Free radiation condition

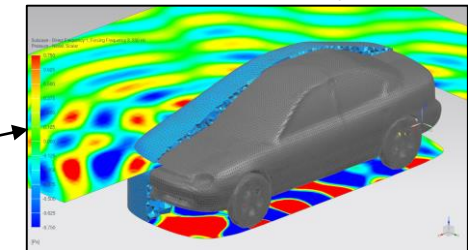


RECEIVER

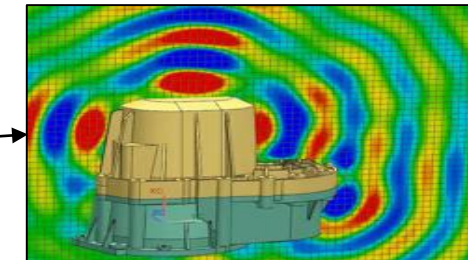
Cabin Powertrain Noise



Exterior Pass-By Noise



Ext. Component Noise

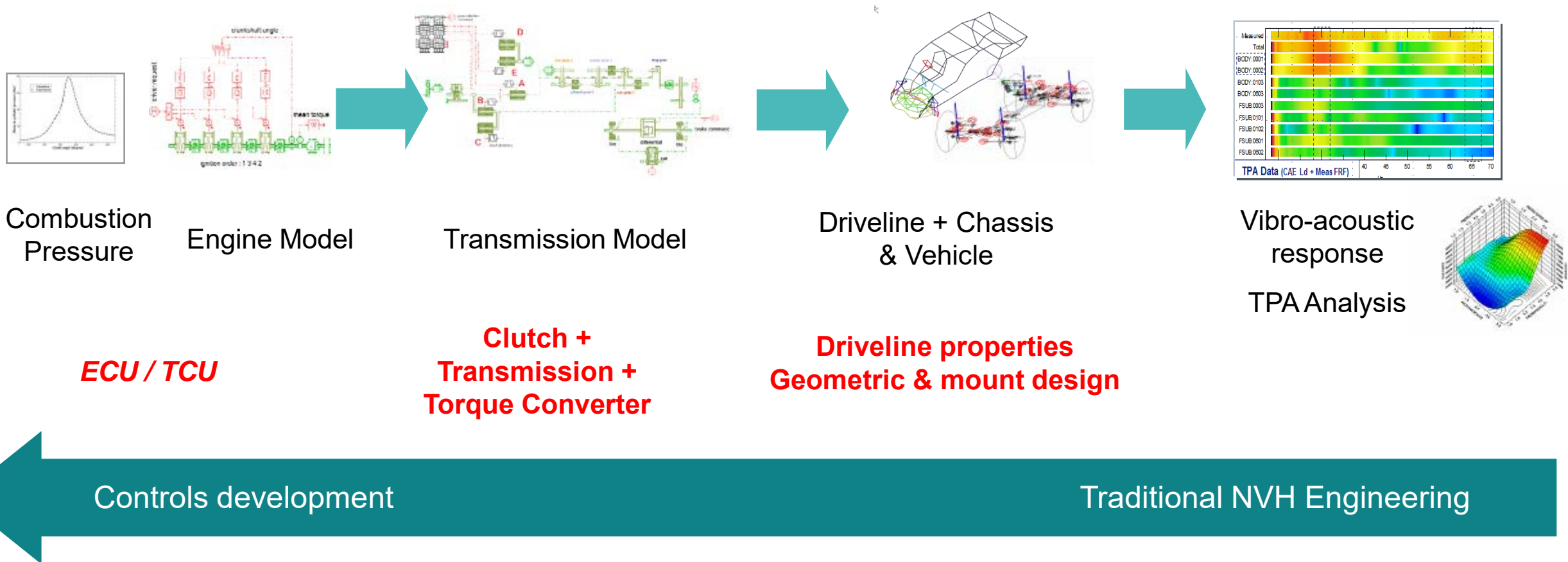


Model-based TPA

Application example: Low frequency driveline booming

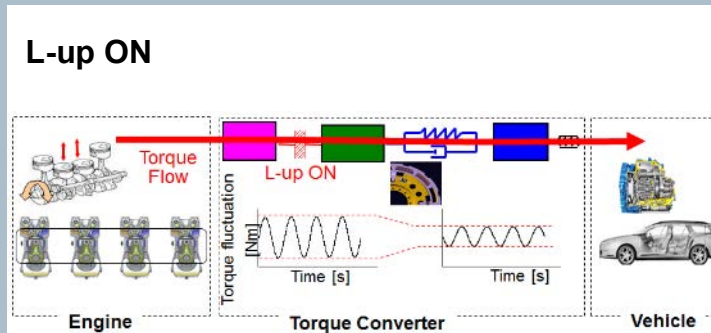


Extending TPA to system simulation and controls development

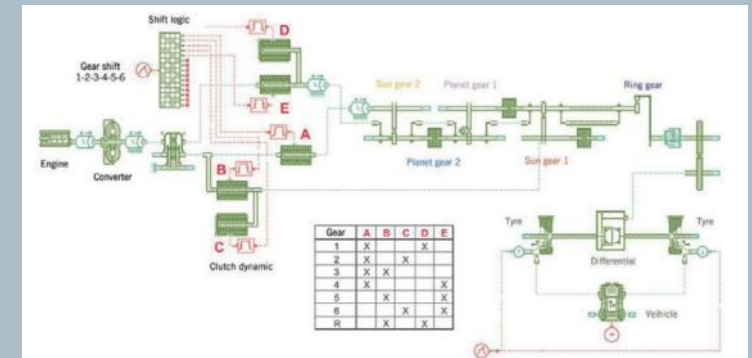




Reducing booming, judder and gear noise



Energy flow lock-up booming



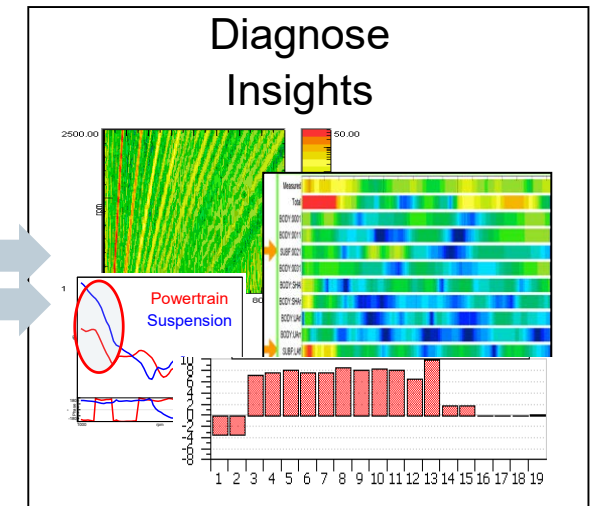
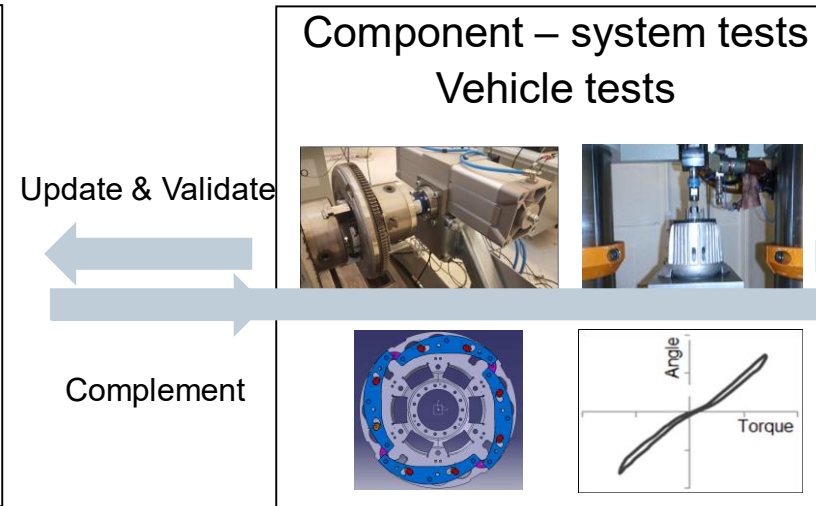
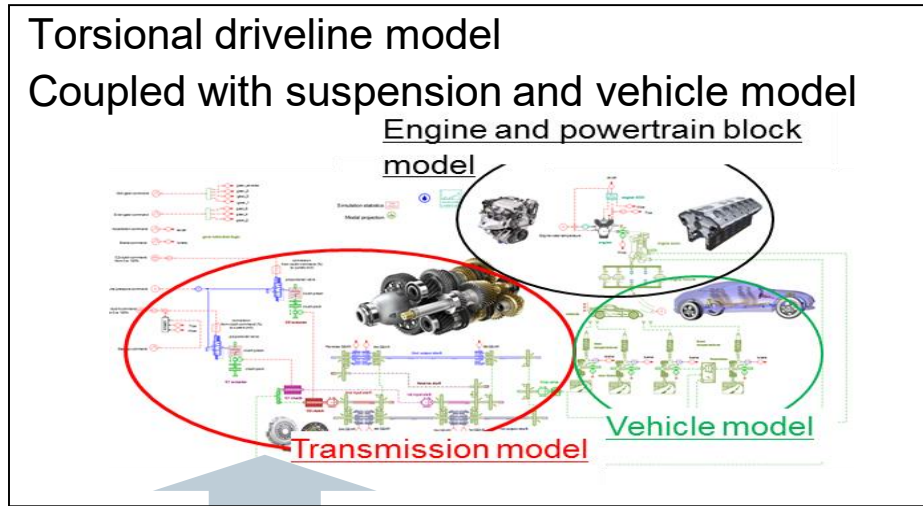
Full-vehicle simulation

- Gained 50 percent time reduction when troubleshooting a new NVH issue
- Significantly reduced overall development time
- Recognized as technology partner of automotive OEMs, resulting in competitive advantage

- Deploy a full vehicle model based approach for the prediction and elimination of clutch judder
- Employ full vehicle modeling approach combining test, 3D and 1D simulation methodologies

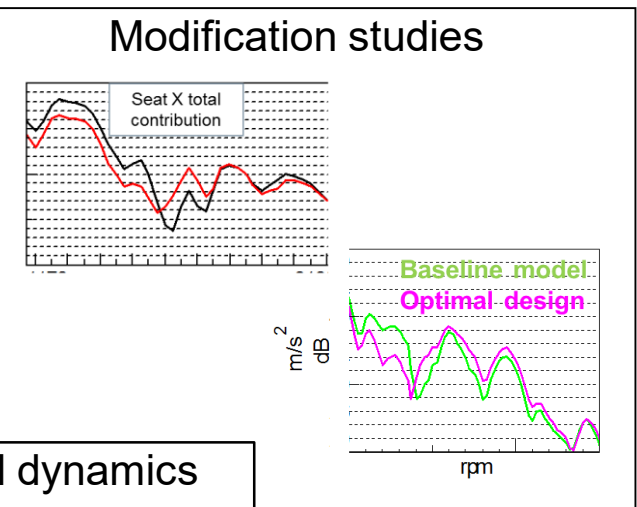
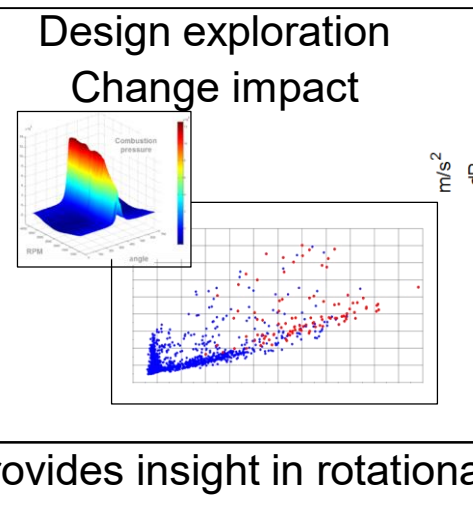
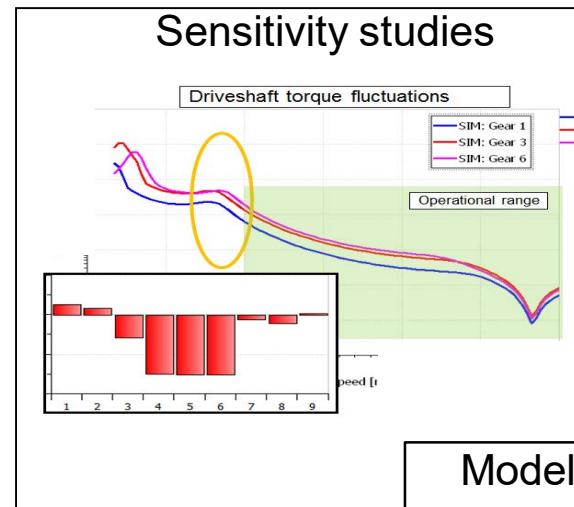
“Many NVH techniques we learned from Simcenter Engineering services are now part of our standard development process, such as transfer path analysis.”

Hiroki Tsuji, Group Manager, Core Component Engineering Department



Scalable model complexity
f(accuracy, information)

	Engine	T/M	SUSPENSION	BODY
Master (not to be modeled)	HF engine	All internal details with clearances	3D full physical	FE
Level 1	MVEM	Simple ratio with equivalent inertia		1D
Level 2	Mapped engine + 3D body	Simple ratio with equivalent inertia and clearance	2D	2D
Level 3	Cylinder pressure tables + 3D body	TM with distributed inertias and stiffness	2D with NVH tire model	2D + FRF



Model provides insight in rotational dynamics & coupling driveline and suspension dynamics

Better & faster vehicle NVH insights using the latest transfer path analysis methods

Introduction to transfer path analysis

Traditional TPA methods

Time-domain TPA

Component-based TPA

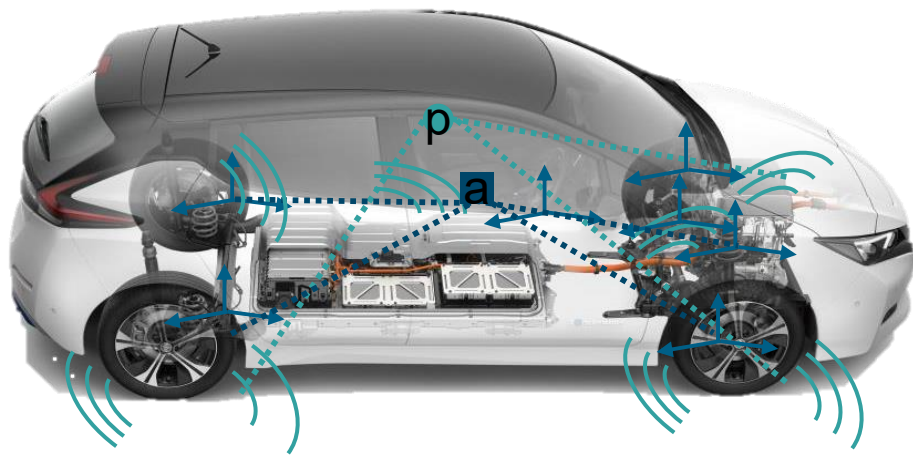
Model-based TPA

Conclusions



Transfer Path Analysis

Conclusion



Transfer Path Analysis (TPA) has been a key method in NVH Engineering for the past decades

Evolutions have kept TPA up-to-date:

- ✓ Faster TPA analyses with tools like OPAX
- ✓ Time-domain TPA to capture transients for eg. start/stop events
- ✓ Using 3D and 1D models to allow contribution analysis throughout development
- ✓ Component-based TPA to extend modular approaches from passive components to include the actual sources

Thank you! Want to know more?

Read more



Explore, share and learn

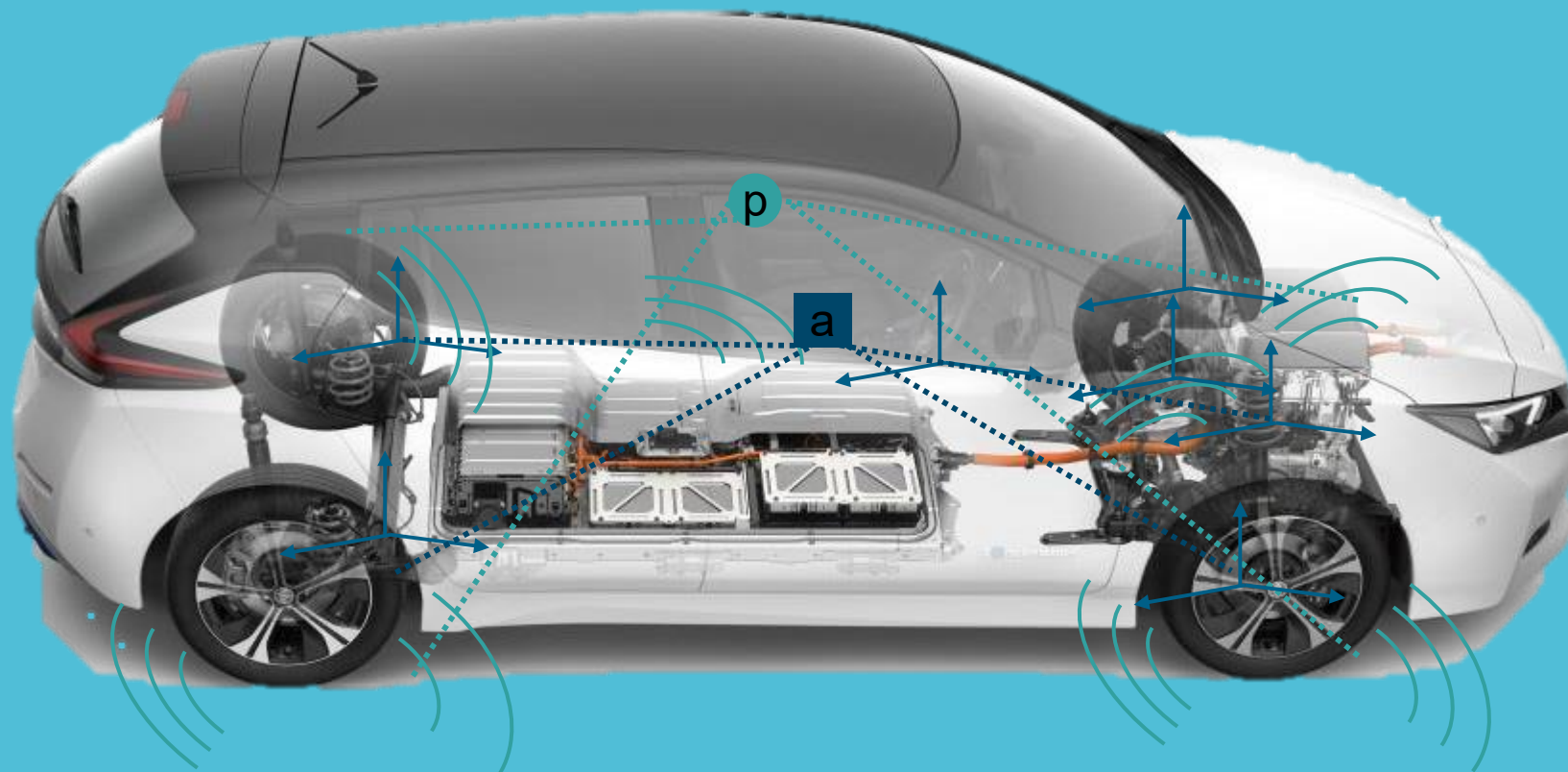


Watch videos



Contact the expert





Thank you!

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BAE Systems	Irkut	Valeo
BASF	Jaguar Landrover	VDL
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